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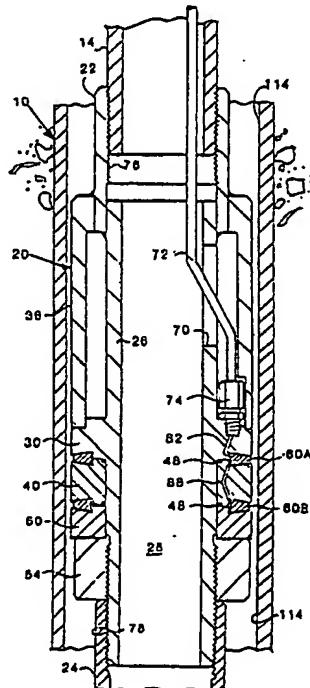
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(54) Title: PACKER ARRANGEMENTS FOR OIL WELLS AND THE LIKE

## (57) Abstract

Packer (20) includes a body member (26), and a metal ring (60, 60') is disposed in its entirety within an annular recess in the body member (26). The recess has opposed die surfaces (38, 42; 44, 52; 38', 42') and a circumferential opening extending entirely around the peripheral outer surface of the body member (26). Hydraulic fluid is flowed through passage (82, 82') into the recess to apply pressure to the inner peripheral surface (62, 62') of the metal ring (60) to expand the ring radially outwardly and force its outer peripheral surface (64, 64') through the circumferential opening and into annular sealing engagement with the opposed surface (114, 114') of the well casing (10, 10').



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-1-

PACKER ARRANGEMENTS FOR OIL WELLS AND THE LIKE

This invention relates to technology for subterranean recovery of oil and the like, and more particularly to seal arrangements for oil wells and the like.

A variety of seal or packer arrangements have been devised to seal petroleum producing wells. A typical seal arrangement is of the baffle type, in which one or more cylinders are placed in longitudinal compression. Typical materials for these types of deformable seals are elastomers and special components such as graphite, asbestos, and wire. In oil recovery enhancement techniques, and in wells used in geothermal production, the packer frequently must seal the casing or tube annulus under conditions of elevated temperature, high pressure and corrosive surface environment. Because of the harsh environment and other technical requirements associated with such applications, a very durable seal that can be deployed with relative ease is necessary.

In accordance with the invention, there is provided a packer that includes a body member with an annular recess in which a metal ring in its entirety is disposed. The recess has an annular opening at the outer surface of the body and has an outer peripheral surface for sealing engagement with the casing wall. Application of hydraulic pressure on the inner peripheral surface of the ring causes radially outward expansion and plastic deformation of the ring and forces the outer peripheral surface of the ring through the annular opening of the recess and into sealing engagement with the opposed surface of the casing or tube in which the packer is disposed.



-2-

In preferred embodiments, the packer body is an elongated tubular casing, and the metal ring is disposed in its entirety within the annular recess in the casing. Hydraulic fluid is flowed into the 5 recess to expand the ring radially outwardly and force its outer peripheral surface through the circumferential recess opening and into annular sealing engagement with the opposed surface of the casing. Juxtaposed die surfaces in the recess are 10 inclined towards one another (preferably at a total die angle of less than fifty degrees) in the radially outward direction. The metal ring itself (or a supplemental ring of greater ductility than the packer seal ring) has a trapezoidal cross sectional 15 shape with surfaces of that ring in sealing engagement with the inclined die surfaces so that application of hydraulic pressure to that ring produces plastic deformation of the trapezoidally shaped sealing ring and at the same time forces a portion 20 of the metal packer seal ring through the circumferential opening. While aluminum and titanium are useful metals, other similar metals may also be utilized. Supplemental sealing enhancement means in the form of surface treatment or added material 25 may be applied to the outer peripheral surface of the metal packer seal ring.

The packer is a compact device which provides a durable and conformable seal that is particularly useful in harsh subterranean environments where high 30 pressures and high temperatures and corrosive conditions are encountered.



-3-

Further features and advantages of the invention will be seen as the following description of particular embodiments progresses, in conjunction with the drawings, in which:

5 Fig. 1 is a diagram of a typical application of a packer in accordance with the invention;

Fig. 2 is a sectional view taken along the line 2-2 of Fig. 1;

10 Fig. 3 is a sectional view, similar to Fig. 2 and on a larger scale, of die components of the packer shown in Fig. 2;

15 Figs. 4 and 5 are diagrammatic views of the packer showing the seal ring in initial condition (Fig. 4) and in deployed condition (Fig. 5);

Fig. 6 is a sectional view, similar to Fig. 2 but on a larger scale, showing a packer ring in deployed condition;

20 Fig. 7 is a graph indicating characteristic packer rings of various materials of configuration similar to that shown in use in the packer shown in Figs. 1 and 2;

Fig. 8 is a sectional view of another packer ring embodiment; and

25 Fig. 9 is a sectional view, similar to Fig. 8, showing the packer ring in deployed condition.

#### Description of Particular Embodiments

Shown in Fig. 1 is a casing 10 that extends downwardly from the surface of the ground to an oil reservoir 12 or other subsurface geologic formation.

30 Disposed within casing 10 is an integrated flowing system that includes tubing sections 14, 16 (and other appropriate hardware not shown) and a packer



-4-

unit 20 that has an upper coupling 22 connected to tubing section 14 and a lower coupling section 24 connected to tubing section 16.

Further details of the packer may be seen with reference to Fig. 2. Casing 10 has an inner diameter of about sixteen centimeters. The packer includes a tubular body 26 of heat treated tool steel (e.g., 4140) that is about thirty-eight centimeters in length and defines a through passage 28 about 6.3 centimeters diameter. Formed integrally with body 26 is an annular flange 30 that has an outer diameter of about fifteen centimeters and an axial height of about 3.2 centimeters. Formed on upper surface 32 is an annular peripheral recess 34 (Fig. 3) which receives sleeve 36; and formed in the lower surface of flange 30 is an inclined annular die surface 38. Seated on flange 30 are series of two die rings 40, 50 that are secured by nut 54 that is threadedly received on the lower end of tube 26. Formed in die ring 40 are two die surfaces 42, 44 each inclined at 7 1/2 degrees. Inclined die surface 52 is similarly formed in ring 50.

A metal ring 60 of trapezoidal cross-sectional configuration is disposed in each recess defined by opposed die surface 38, 42 and 44, 52. Each ring has a width or radial dimension of 1.75 centimeter, a height at its inner peripheral surface 62 of 1.25 centimeter, a height at its outer peripheral surface 64 of 0.75 centimeters. The upper and lower surfaces 66, 68 of each ring 60 are clamped firmly between the opposed extruding die surfaces 38, 42 and 44, 52 when the die rings 40, 50 are clamped against die flange 30 by nut 54.



-5-

Body 26 has an aperture 70 at its upper end through which extends a length of 0.65 centimeter stainless steel tubing 72 that has a burst strength of about 40,000 psi and that is connected by fitting

5 74 to flow passage 82 formed in flange 30.

Coupling 22 is welded to body 26 and has internal threads 76 that engage tubing section 14 while the lower end of body 26 has external threads 78 on which coupling 24 is received.

10 Further details of die flange 30 and die rings 40, 50 are shown in Fig. 3. Threaded hole 80 in upper surface 32 of flange 30 receives coupling 74. Passage 82 extends to an outlet in cylindrical die recess surface 84 (which has a diameter of about

15 10.8 centimeters). Extrusion surface 38 is inclined downwardly in an outward direction at an angle of 7 1/2 degrees and extends to lip 86. Ring 40 has similar annular axially extending surfaces 90, 92 in which are formed ports of through passage 88, and 20 similar annular extrusion lips 94, 96. Ring 50 has a similar annular wall 100 at the inner edge of inclined extrusion surface 52 and an annular lip 102 at the outer periphery of extrusion surface 52.

With the extrusion rings 40 and 50 clamped against 25 flange 30, seals 48 in grooves 46 seal the joints between the rings and two annular recesses of trapezoidal configuration are defined, each of the same shape and each of about 1.9 centimeter radial length. Disposed in each recess is a metal ring 60,

30 the inclined surfaces 66, 68 of which are firmly clamped and sealed against the upper and lower extruding surfaces of the recess.



-6-

Hydraulic fluid applied through line 72 and coupling 74 flows through passage 82 and acts against the inner surface 62 of the upper ring 60A and through passage 88 to the lower recess and 5 similarly acts against surface 62 of ring 60B, as indicated by arrows 110. As the fluid pressure in the radial outward direction on the back faces 62 of the rings increases, each ring is deformed first elastically and then plastically as the ring is 10 forced through the lips of each die in an extrusion action creating a ring of greater diameter and surface area, with an extruded seal ring portion 112 and with outer surface 64 seated against the inner surface 114 of casing 10 as indicated in Figs. 5 and 6.

15 The required fluid pressure is a function of the die angle, the necessary radial extension, the ring material, and the ring geometry. Shown in Fig. 7 is a graph indicating the required fluid pressure for aluminum and titanium ring materials as 20 a function of total die angle. The packer is designed to withstand a maximum temperature of 343 degrees C with a useful life as a function of the differential pressure it supports. Where potentially large pressure induced forces act on the seal ring portion 112 at maximum temperatures and for long 25 life situations (greater than one year), a packer fluid 116 can be placed on top of the seal ring 112 to reduce or eliminate the pressure differential across the seal and greatly reduce metal creep 30 effects. The hydraulic pressure may be increased if desired by the incorporation in the packer 20 of differential area piston intensifiers between supply tube 72 and passages 82. Sealing efficiency can be



-7-

modified by changing ring geometry, for example, by altering sealing area or changing the outer edge configuration of the ring. The addition of serration 120 to surface 64, or other surface treatments 5 allows the ring to seal effectively against irregular casing surfaces. By altering the initial ring geometry, the die angle, or the ring thickness, the effective seal area can be adjusted, and the deployed seal ring 112 can be strengthened by work hardening 10 effects. Appropriate preventive means can be used to minimize environmental effects, such as ring corrosion.

The packer arrangement is a compact device with a minimum of components which provides a durable and 15 conformable seal that is particularly useful in harsh environments. Another seal arrangement is shown in Figs. 8 and 9. Similar components in that figure have been identified with primed reference numerals, for example, casing 10'. The seal ring 60' is a 20 composite of a titanium ring 130 with parallel upper and lower faces 132, 134 disposed in sliding fit between parallel radially extending zero angle die surfaces; and a heat shrunk aluminum ring 136 disposed in an annular recess in the outer peripheral surface 25 64'. Aluminum insert 136 carries serrations 138 that projects beyond the outer face 64' of the titanium ring 130. An inner ring 140 of aluminum (1100-0) of trapezoidal cross-section is positioned inside titanium ring 130 with the outer peripheral surface 30 142 of ring 140 abutting the inner peripheral surface of ring 130. The upper and lower inclined surfaces of ring 140 are seated against extrusion surfaces 38', and 42' of the die members.



-8-

Application of hydraulic pressure through line passage 82' acts against the inner surface 62' of aluminum ring 140, forcing that ring outwardly in an extrusion process which acts against the inner 5 surface of titanium ring 130. The titanium ring is expanded radially through the "zero angle" die to the position shown in Fig. 9 with the aluminum seal 136 seated against wall 114' of casing 10' as shown in Fig. 9.

10 While particular embodiments of the invention have been shown and described, various modifications will be apparent to those skilled in the art, and therefore it is not intended that the invention be limited to the disclosed embodiments or to details 15 thereof and departures may be made therefrom within the spirit and scope of the invention.

What is claimed is:



-9-

Claims

1. A packer for use in an oil well system or  
the like comprising

5 a body for disposition in an elongated tubular  
casing,

characterized by the provision of an annular  
recess in said body, said recess having a circum-  
ferential opening that extends entirely around the  
peripheral outer surface of said body,

10 a metal ring disposed entirely within said  
recess, said ring being of ductile material and  
having an outer peripheral surface for sealing  
engagement with the casing wall and an inner  
peripheral surface, and

15 a port in said recess for flowing hydraulic  
fluid into said recess to apply pressure to the  
inner peripheral surface of said metal ring to  
expand said ring radially outwardly and force said  
outer peripheral surface through said circumferential  
20 opening and into annular sealing engagement with the  
opposed surface of said casing.

2. The packer of claim 1 further characterized  
by the provision of juxtaposed die surfaces in said  
recess, said die surfaces being inclined towards one  
25 another in the radially outward direction towards  
said circumferential opening, said ring having a  
trapezoidal cross sectional shape with surfaces of  
said ring in sealing engagement with said die sur-  
faces so that application of hydraulic pressure to  
30 said ring produces plastic deformation of said ring  
and extrusion of a portion of said metal ring through  
said circumferential opening.



-10-

3. The packer of claim 2 further characterized in that the total included angle of said juxtaposed die surfaces is less than fifty degrees.

4. The packer of any preceding claim further  
5 characterized in that said body is of elongated  
tubular configuration with a cylindrical outer sur-  
face in which said annular opening is formed, and  
has coupling structure at each end of said body for  
attachment to a tube string or the like.

10 5. The packer of any preceding claim further  
characterized by the provision of means on the outer  
peripheral surface of said ring for enhancing the  
effectiveness of the seal against said casing surface.

15 6. The packer of any preceding claim further  
characterized in that the metal of said ring is  
selected from aluminum and titanium and alloys  
thereof.

20 7. The packer of any preceding claim further  
characterized in that the radial cross sectional  
width of said metal ring is greater than the space  
between the outer surface of said body and the inner  
surface of said casing, and the radial cross  
sectional width of said recess is greater than the  
radial cross sectional width of said metal ring.



-11-

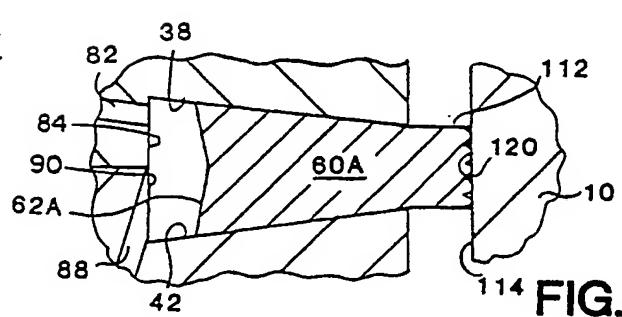
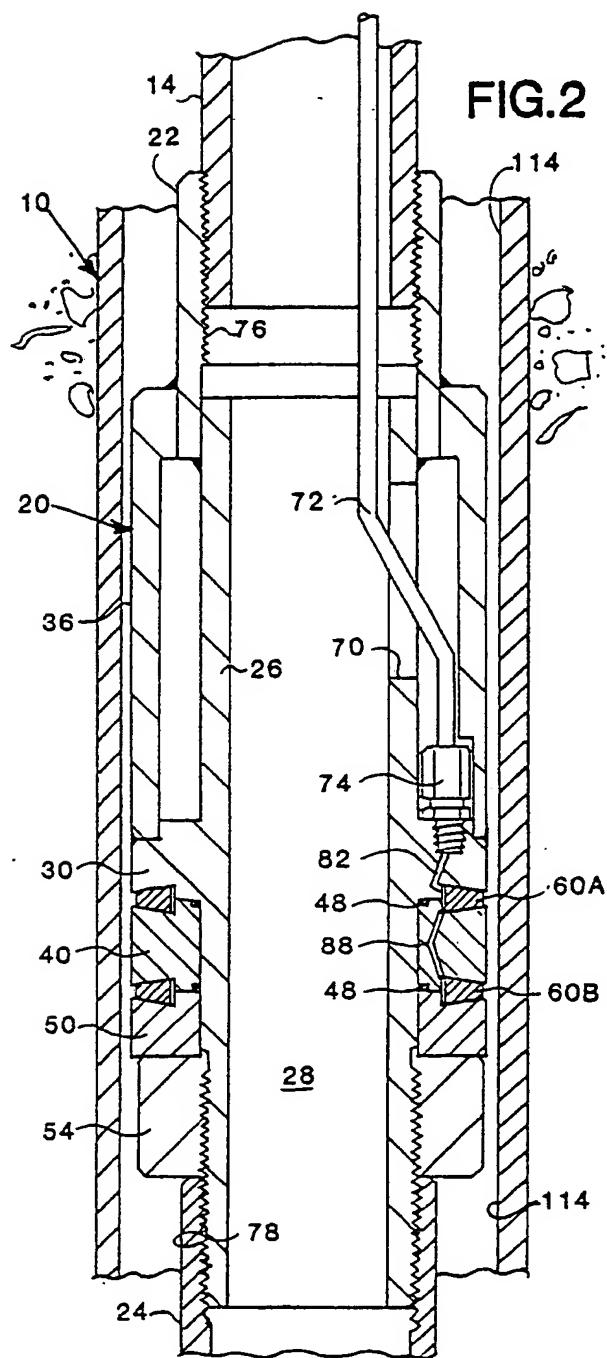
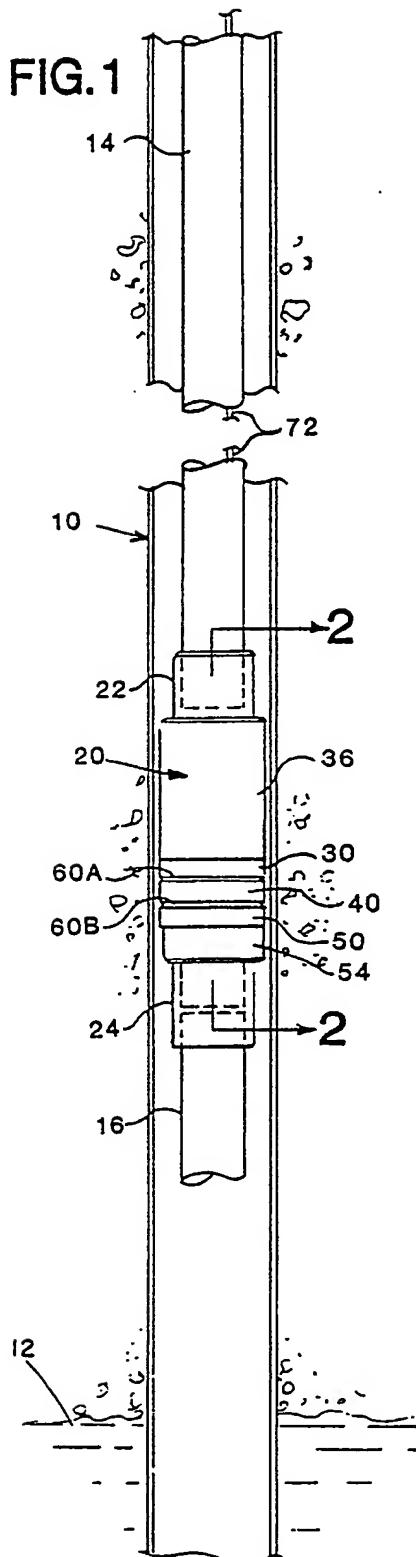
8. The packer of any preceding claim further characterized by the provision of a supplemental ring of more ductile material than said metal ring, said supplemental ring being disposed in said recess 5 inside of said metal ring.

9. The packer of any of claims 1-7 further characterized in that said ring includes an inner annular component of soft metal and an outer annular component of harder metal.

10 10. The packer of claim 9 further characterized in that said inner annular component is of trapezoidal cross sectional shape and said outer annular component is of rectangular cross sectional shape.

11. The packer of claim 10 further characterized 15 in that said inner annular component is of aluminum and said outer annular component is of titanium.





14 FIG. 6



FIG.3

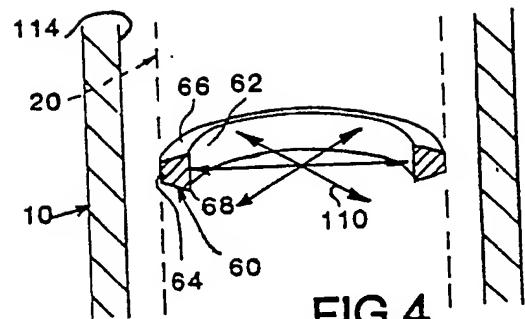
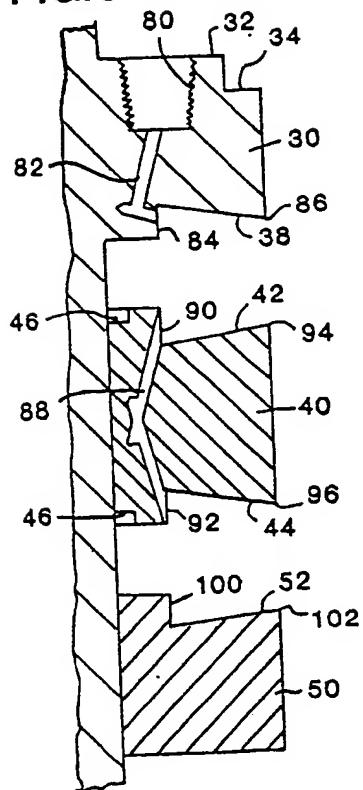


FIG.4

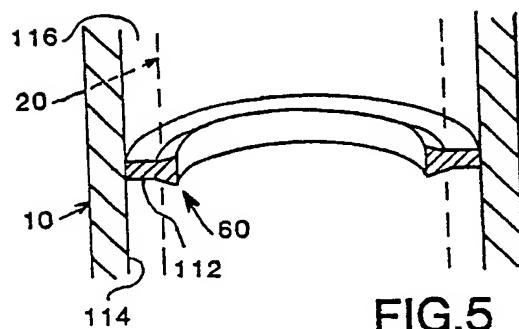


FIG.5

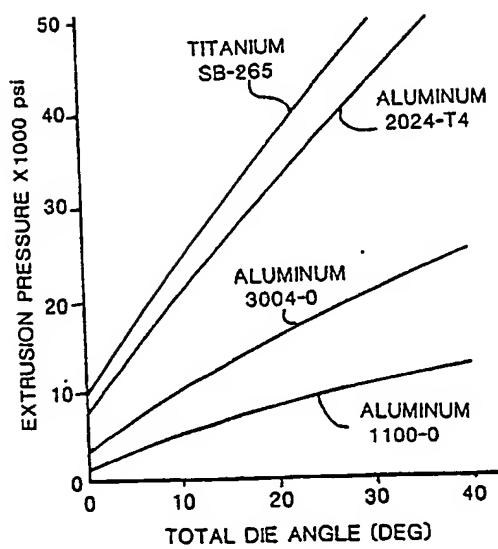


FIG.7

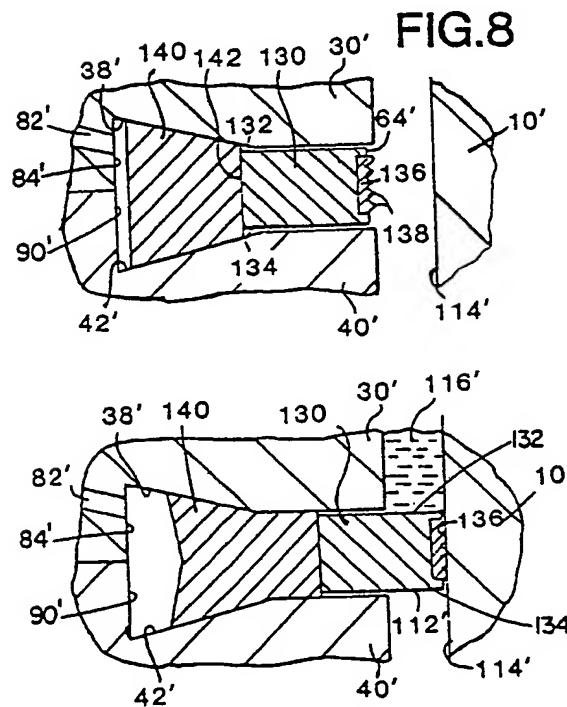


FIG.9



## INTERNATIONAL SEARCH REPORT

International Application No PCT/US81/00216

I. CLASSIFICATION OF SUBJECT MATTER (If several classification symbols apply, indicate all) <sup>3</sup>

According to International Patent Classification (IPC) or to both National Classification and IPC

INT. CL. F16J 15/46  
U.S. CL. 277/27, 34, 73, 236

## II. FIELDS SEARCHED

Minimum Documentation Searched <sup>4</sup>

Classification System	Classification Symbols
U.S.	277/3, 27, 34, 34.3, 34.6, 73, 188A, 226, 236
Documentation Searched other than Minimum Documentation to the Extent that such Documents are Included in the Fields Searched <sup>5</sup>	

III. DOCUMENTS CONSIDERED TO BE RELEVANT <sup>14</sup>

Category <sup>6</sup>	Citation of Document, <sup>16</sup> with indication, where appropriate, of the relevant passages <sup>17</sup>	Relevant to Claim No. <sup>18</sup>
A	US A 553,738 Published 28 January 1896 Raynsford	1-11
A	US A 2,192,805 Published 5 March 1940 Seemark	1-11
A	US A 2,237,709 Published 8 April 1941 Lowe	1-11
A	US A 2,529,744 Published 14 November 1950 Schweitzer	1-11
A	US A 3,298,719 Published 17 January 1967 Bills et al	1-11
A	US A 4,178,020 Published 11 December 1979	1-11
A, E	US A 4,256,314 Published 17 March 1981	1-11
A	GB A 186,594 Published 1924 Metropolitan, ETC.	1-11

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"X" document of particular relevance

## IV. CERTIFICATION

Date of the Actual Completion of the International Search <sup>2</sup>

04 June 1981

Date of Mailing of this International Search Report <sup>3</sup>

19 JUN 1981

International Searching Authority <sup>1</sup>

ISA/US

Signature of Authorized Officer <sup>30</sup>*Robert L. Ward, Jr.*

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